

of Prandtl

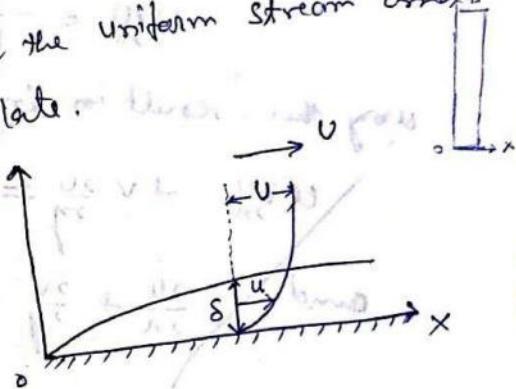
## Flow parallel to a semi-infinite flat plate (The Blasius soln)

Consider the <sup>Steady</sup> flow of an incompressible viscous fluid past a thin flat plate which is placed in the direction of a uniform stream of velocity  $U_0$ . The plate is of finite length. Hence problem is one of 2-dim <sup>friction much larger breadth</sup> motion which can be analyzed by using the ~~Prandtl~~ boundary layer equations. Let the origin of the co-ordinates be at the leading edge of the plate, the  $x$ -axis be the direction of the uniform stream and the  $y$ -axis normal to the plate.

The Prandtl boundary layer Equations in the case under consideration are

$$(1) \quad u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \frac{\partial^2 u}{\partial y^2}$$

$$(2) \quad \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$



$$\left. \begin{array}{l} (1) \\ (2) \end{array} \right\}$$



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outside the B.L. we can approximate the flow as non-viscous fluid flow, thus we have by Euler's eqn of motion

$$\frac{DU}{Dt} = -\frac{1}{\rho} \frac{\partial p}{\partial x}$$

$$\frac{\partial U}{\partial t} + U \frac{\partial U}{\partial x} = -\frac{1}{\rho} \frac{\partial p}{\partial x} \quad \text{--- (1)}$$

for steady flow

$$U \frac{\partial U}{\partial x} = -\frac{1}{\rho} \frac{\partial p}{\partial x} \quad \text{--- (2)}$$

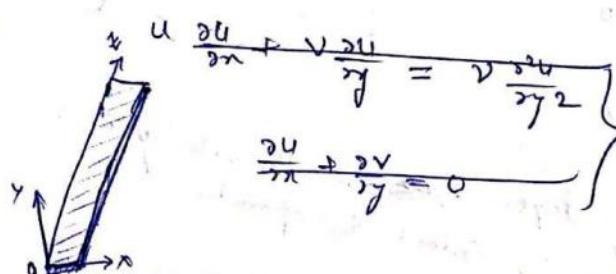
now this result is Bernoulli Eqn becomes

$$U \frac{\partial U}{\partial x} + V \frac{\partial U}{\partial y} = U \frac{\partial U}{\partial x} + V \frac{\partial U}{\partial y} \quad \text{--- (3)}$$

$$\text{and } \frac{\partial U}{\partial x} + \frac{\partial V}{\partial y} = 0$$

If  $U$  is constant (for uniform stream)

Above (3). Bernoulli reduces to



$$\frac{\partial p}{\partial x} = 0$$

$p = \text{const.}$  outside the B.L. and thus approximately uniform throughout the B.L. Hence

for steady flow over a thin flat plate B-L Eqn reduces to

$$U \frac{\partial U}{\partial x} + V \frac{\partial U}{\partial y} = V \frac{\partial U}{\partial y}$$



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∴ by (1) reduces to

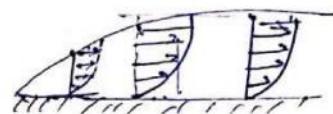
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$$\begin{aligned} u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} &= \nu \frac{\partial^2 u}{\partial y^2} - (2) \\ \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} &= 0 \end{aligned} \quad \left. \begin{array}{l} \\ \end{array} \right\} - (3)$$

with the B.C.

$$u = v = 0 \quad ; \quad y = 0 \quad - (i)$$

$$u = U \quad ; \quad y = \infty \quad - (ii)$$



we assume velocity profile as

$$u = U \cdot F(x, y, v, U) = U \cdot F(\eta) \quad - (4) \quad \begin{array}{l} (\text{similarity soln.}) \\ \eta - \text{similarity variable} \end{array}$$

where  $\eta$  is dimensionless parameter. Since the B.C. profile will vary only in scale and will be fixed in shape so obvious  
choice of parameter  $\eta = \frac{y}{\delta} = \frac{y}{\sqrt{\frac{vx}{U}}} = y x^{-\frac{1}{2}} \sqrt{\frac{U}{v}}$  — (5)  
 $\therefore o(\delta) = o\left(\frac{vx}{U}\right)$

If  $\psi$  is stream function then we can take

$$u = \frac{\partial \psi}{\partial y} \quad \& \quad v = -\frac{\partial \psi}{\partial x}$$

$$\psi = \int u dy = \int_U F(\eta) \cdot \frac{\partial \eta}{\partial y} d\eta = U \int F(\eta) \cdot \sqrt{\frac{xy}{U}} d\eta \quad (\text{by eqns. 5})$$

$$\psi = \sqrt{Uvx} \int F(\eta) d\eta = \sqrt{Uvx} \cdot f(\eta) \quad - (6)$$

$$\text{where, } f(\eta) = \int F(\eta) d\eta$$



$$u = U F(\eta) = U f(\eta)$$

$$\text{and } v = -\frac{\partial u}{\partial x} = -\sqrt{Uv}x \cdot f'(\eta) \cdot \frac{m}{2n} - \sqrt{Uv} f(\eta) \cdot \left(\frac{1}{2} \frac{1}{\sqrt{n}}\right)$$

$$= -\sqrt{Uv}x \cdot f'(\eta) \cdot 2\sqrt{\frac{U}{V}} \left(-\frac{1}{2} \cdot \frac{1}{\sqrt{n}}\right) - \frac{1}{2} \sqrt{\frac{Uv}{n}} f(\eta)$$

$$= +\sqrt{Uv} \left(\frac{n}{2x}\right) f'(\eta) - \frac{1}{2} \sqrt{\frac{Uv}{n}} f(\eta)$$

$$= \frac{1}{2} \sqrt{\frac{Uv}{n}} \left[n f'(\eta) - f(\eta)\right]$$

we can evaluate

$$\frac{\partial u}{\partial n} = \frac{\partial^2 u}{\partial x \partial y} = -\frac{U}{2} \frac{n}{x} f''(\eta)$$

$$\frac{\partial u}{\partial y} = U \sqrt{\frac{U}{v n}} f''(\eta)$$

$$\frac{\partial u}{\partial x} = \frac{U^2}{v n} f'''(\eta)$$

Substituting  $u, v, u_x, u_y$  &  $u_{yy}$  in eq<sup>2</sup> we get

$$-\frac{U^2}{2} \frac{n}{x} f' f'' + \frac{U^2}{2n} (\eta f' - f) f''' = \frac{U^2}{n} f'''$$

$$\text{i.e. } 2f''' + f \cdot f'' = 0 \quad \text{--- (7)}$$

the B.C. transform into in terms of  $f \circ \eta$  are

$$\text{the B.C. transform into in terms of } f \circ \eta \text{ are}$$

$$(i) \quad \eta = 0 : \quad f = 0, \quad \frac{df}{dn} = 0$$

$$(ii) \quad \eta \rightarrow \infty : \quad \frac{df}{dn} \rightarrow 1$$

$\Sigma^m$  is usually referred as the Martin's Eq<sup>m</sup>.



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$\text{Eqm}$  is 3<sup>rd</sup> order nonlinear diff. eqn and no closed form sol<sup>n</sup> has been found. Bôcher in 1908 obtain the sol<sup>n</sup> in the form of power series about  $\eta=0$ .

$$f = A_0 + A_1 \eta + \frac{A_2}{2!} \eta^2 + \frac{A_3}{3!} \eta^3 + \dots$$

$$f' = A_1 + A_2 \eta + \frac{A_3}{2!} \eta^2 + \frac{A_4}{3!} \eta^3 + \dots$$

$$f'' = A_2 + A_3 \eta + \frac{A_4}{2!} \eta^2 + \frac{A_5}{3!} \eta^3 + \dots$$

$$f''' = A_3 + A_4 \eta + \frac{A_5}{2!} \eta^2 + \frac{A_6}{3!} \eta^3 + \dots$$

using B.C. (i) at  $\eta=0$  to eqn for  $f$  &  $f'$ . we get

$$A_0 = 0 ; A_1 = 0$$

Substituting the result value thus obtained for  $f, f'', f'''$

in (7) we find

$$2A_3 + (2A_4)\eta + (A_2^2 + 2A_5)\frac{\eta^2}{2!} + (4A_2A_3 + 2A_6)\frac{\eta^3}{3!} + \dots = 0$$

Evaluating coefficient both side

$$\text{we get } A_3 = A_4 = A_6 = A_7 = 0$$

$$A_5 = -\frac{A_2^2}{2}$$

$$A_8 = \frac{11}{4} A_2^3 \quad \dots$$

$$\text{So, } f = \frac{A_2}{2!} \eta^2 - \frac{1}{2} \frac{A_2^2}{5!} \eta^5 + \frac{1}{4} \frac{11}{8!} A_2^3 \eta^8 - \frac{1}{8} \cdot \frac{375}{11!} A_2^4 \eta^{11} \quad \dots$$

Solved eqn satisfies B.C (i) at  $\eta=0$  and constant  $A_2$



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will be determined from the B.C. at  $\eta = \infty$

(5)

Expression for  $f$  can be written as

$$f = A_2^{\frac{1}{2}} \left[ \frac{(A_2^{\frac{1}{2}}\eta)^2}{2!} - \frac{1}{2} \frac{(A_2^{\frac{1}{2}}\eta)^5}{5!} + \frac{1}{4} \cdot \frac{11}{8!} (A_2^{\frac{1}{2}}\cdot\eta)^8 - \dots \right]$$

$$f(\eta) = A_2^{\frac{1}{2}} \cdot G(A_2^{\frac{1}{2}}\cdot\eta)$$

now using B.C. at  $\eta = \infty$

$$\lim_{\eta \rightarrow \infty} f'(\eta) = A_2^{\frac{3}{2}} \cdot G'(A_2^{\frac{1}{2}}\eta)$$

$$= A_2^{\frac{3}{2}} \cdot \lim_{\eta \rightarrow \infty} G'(\eta)$$

$$\therefore A_2 = \left[ \lim_{\eta \rightarrow \infty} \frac{1}{G'(\eta)} \right]^{\frac{2}{3}}$$

we can determine  $A_2$  numerically ~~or~~ which is

$$A_2 = 0.33206$$

Also we find  $f''(0) = A_2 = 0.33206$

Shearing stress: at the Plate:

$$\tau_0 = \left( \mu \frac{\partial u}{\partial y} \right)_{y=0} = \left( \mu \frac{\partial u}{\partial y} \right)_{\eta=0}$$

$$= \mu \left( U f''(\eta) \sqrt{\frac{U}{Vx}} \right)_{\eta=0}$$

$$= \mu U f''(0) \sqrt{\frac{U}{Vx}} = \underbrace{0.33206 \mu U \sqrt{\frac{U}{Vx}}}_{\text{Ansatz}}$$

$$= 0.33206 \rho V U \sqrt{\frac{U}{Vx}} = 0.33206 f U^2 \sqrt{\frac{U}{Vx}}$$



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## Boundary-Layer characteristics :

So far we have introduced only one length,  $\delta$ , which is characteristic of the boundary-layer thickness. This 'total boundary-layer thickness' is, however, a nebulous quantity in some respects. For this reason and also some other reasons, it is usual to define a small number of lengths, each is a characteristic of the boundary-layer thickness in some way.

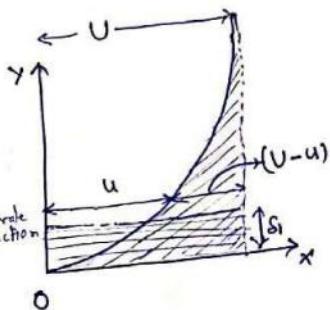
### (ii) Displacement thickness ( $\delta_1$ ):

Displacement thickness is defined as

$$U \cdot \delta_1 = \int_0^\infty (V - u) dy \quad (1)$$

decrease in flow rate due to velocity reduction within the B.L.

$$\delta_1 = \int_0^\infty \left(1 - \frac{u}{U}\right) dy$$



R.H.S. of (1) is decrease in total flow rate caused by the action of the friction and the L.H.S. represents the flow rate in absence of viscosity, i.e. the flow that has been displaced from the wall.

### (iii) Momentum thickness ( $\delta_2$ ): It is defined as

$$\rho U^2 \cdot \delta_2 = \rho \int_0^\infty u (V - u) dy \quad (2)$$

$$\delta_2 = \int_0^\infty \frac{u}{U} \left(1 - \frac{u}{U}\right) dy$$



K.H.S of (2) represents the loss of momentum flow ~~as~~ due to the wall friction in the boundary layer and K.H.S of (2) represents the momentum flow in the absence of the boundary layer.

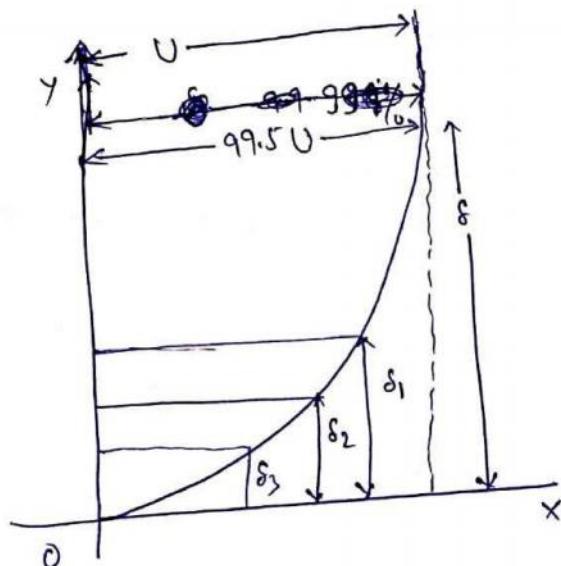
### i) Energy thickness (or K.E. thickness) : ( $\delta_3$ )

It measures the flux of kinetic energy loss within the boundary layer as compared with an inviscid flow.

It is defined as

$$\left(\frac{1}{2} \int u^2\right) \delta_3 = \frac{1}{2} \int_0^\infty (U^2 - u^2) dy$$

$$\delta_3 = \int_0^\infty \frac{u}{U} \left(1 - \frac{u^2}{U^2}\right) dy$$



Comparison of various thickness of a boundary layer

